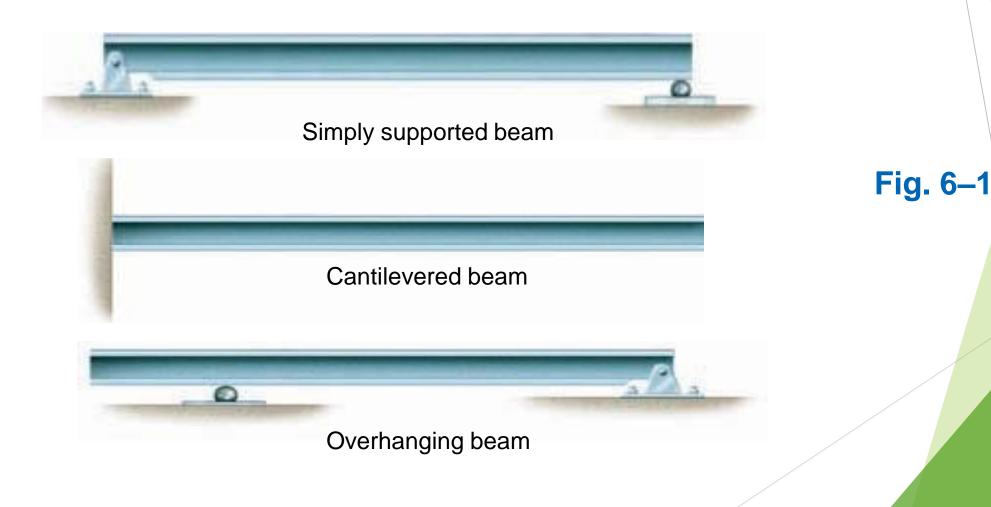
Shear and Moment Diagram

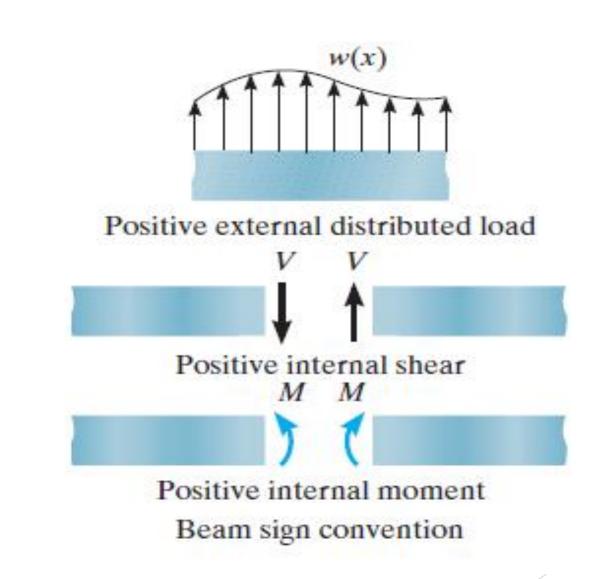
Members that are slender and support loadings that are applied perpendicular to their longitudinal axis are called beams. In general, beams are long, straight bars having a constant cross-sectional area. Often they are classified as to how they are supported. For example, a simply supported beam is pinned at one end and roller supported at the other, Fig. 6-1, a cantilevered beam is fixed at one end and free at the other, and an overhanging beam has one or both of its ends freely extended over the supports.

Beams are considered among the most important of all structural elements. They are used to support the floor of a building, the deck of a bridge, or the wing of an aircraft. Also, the axle of an automobile, the boom of a crane, even many of the bones of the body act as beams.



Beam Sign Convention.

Fig. 6-3



Procedure for Analysis

The shear and moment diagrams for a beam can be constructed using the following procedure.

Support Reactions.

• Determine all the reactive forces and couple moments acting on the beam, and resolve all the forces into components acting perpendicular and parallel to the beam's axis.

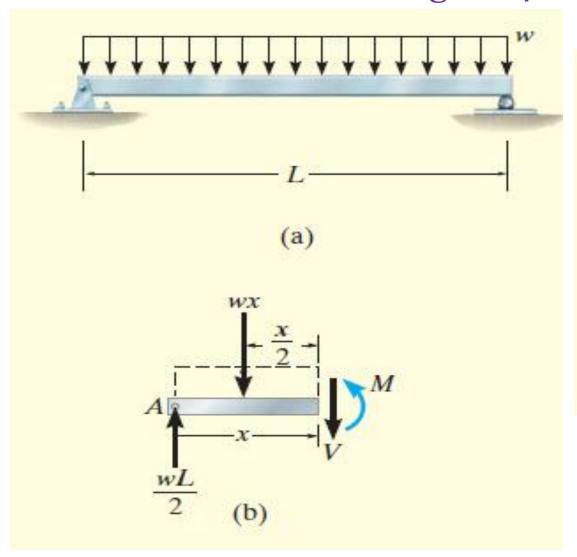
Shear and Moment Functions.

- Specify separate coordinates x having an origin at the bean left end and extending to regions of the beam between concentrated forces and/or couple moments, or where there is no discontinuity of distributed loading.
- Section the beam at each distance x, and draw the free-body diagram of one of the segments. Be sure V and M are shown acting in their positive sense, in accordance with the sign convention given in Fig. 6–3.
- The shear is obtained by summing forces perpendicular to the beam's axis.
- To eliminate V, the moment is obtained directly by summing moments about the sectioned end of the segment.

Shear and Moment Diagrams.

- Plot the shear diagram (V versus x) and the moment diagram (M versus x). If numerical values of the functions describing V and M are positive, the values are plotted above the x axis, whereas negative values are plotted below the axis.
- Generally it is convenient to show the shear and moment diagrams below the free-body diagram of the beam. can be observed that the tangential stress is twice that of the longitudinal stress.

Example 1 Draw the shear and moment diagrams for the beam shown in Fig. 6–4a.



$$+ \uparrow \Sigma F_y = 0; \qquad \frac{wL}{2} - wx - V = 0$$

$$V = w \left(\frac{L}{2} - x\right) \tag{1}$$

$$M = \frac{w}{2}(Lx - x^2) \tag{2}$$

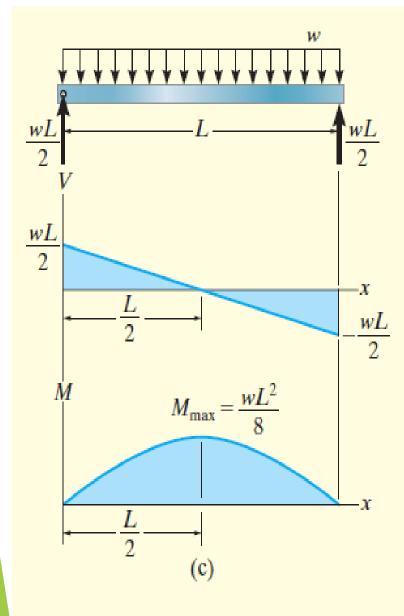


Fig. 6-4

Shear and Moment Diagrams. The shear and moment diagrams shown in Fig. 6–4c are obtained by plotting Eqs. 1 and 2. The point of zero shear can be found from Eq. 1:

$$V = w \bigg(\frac{L}{2} - x \bigg) = 0$$

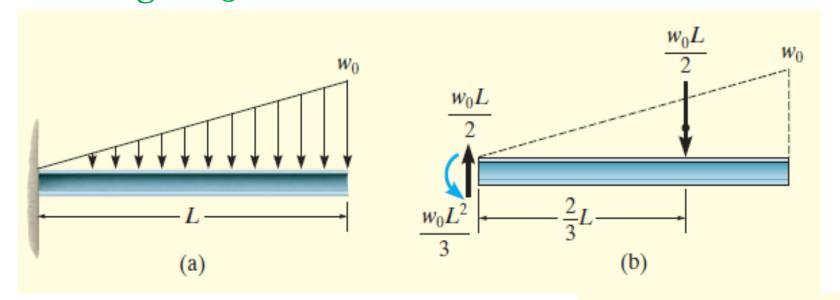
$$x = \frac{L}{2}$$

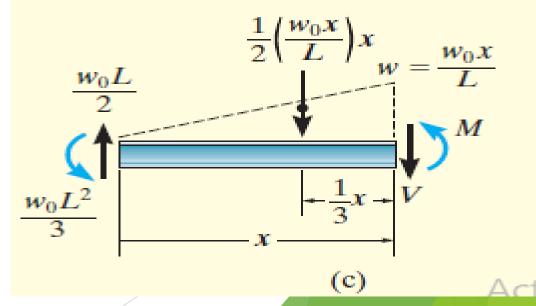
NOTE: From the moment diagram, this value of x represents the point on the beam where the *maximum moment* occurs, since by Eq. 6–2 (see Sec. 6.2) the *slope* V = dM/dx = 0. From Eq. 2, we have

$$M_{\text{max}} = \frac{w}{2} \left[L \left(\frac{L}{2} \right) - \left(\frac{L}{2} \right)^2 \right]$$

$$=\frac{wL^2}{8}$$

Example 2: Draw the shear and moment diagrams for the beam shown in Fig. 6–5a.





$$+\uparrow \Sigma F_y = 0;$$
 $\frac{w_0 L}{2} - \frac{1}{2} \left(\frac{w_0 x}{L}\right) x - V = 0$

$$V = \frac{w_0}{2L}(L^2 - x^2) \tag{1}$$

$$\zeta + \sum M = 0;$$

$$\frac{w_0 L^2}{3} - \frac{w_0 L}{2}(x) + \frac{1}{2} \left(\frac{w_0 x}{L}\right) x \left(\frac{1}{3}x\right) + M = 0 \quad \frac{w_0 L^2}{2}$$

$$M = \frac{w_0}{6L}(-2L^3 + 3L^2x - x^3) \tag{2}$$

These results can be checked by applying Eqs. 6–1 and 6–2 of Sec. 6.2, that is,

$$w = \frac{dV}{dx} = \frac{w_0}{2L}(0 - 2x) = -\frac{w_0x}{L}$$
 OK

$$V = \frac{dM}{dx} = \frac{w_0}{6L}(0 + 3L^2 - 3x^2) = \frac{w_0}{2L}(L^2 - x^2)$$
 OK $-\frac{w_0L^2}{3}$

